# Engineering Software 

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## Erıgirneerirıg Aシショurupijors

When dealing with energy conversion and considering ideal（isentropic）operation and the working fluid is air， the following assumptions are valid：

## Compressible Flow

Single species consideration
Basic equations hold（continuity，momentum and energy equations）
Specific heat is constant

## Eョasic Engine=ring Efuations

## Basic Conservation Equations

## Continuity Equation $\mathrm{m}=\rho \mathrm{vA}[\mathrm{kg} / \mathrm{s}]$

Momentum Equation
F = (vm +pA$)_{\text {out }- \text { in }}[\mathrm{N}]$
Energy Equation

$$
q-w=\left(h+v^{2} / 2+g h\right)_{\text {out }- \text { in }}[k J / k g]
$$

## EEsic Engineering Ecuations

## Ideal Gas State Equation $\mathrm{pv}=\mathrm{RT}[\mathrm{kJ} / \mathrm{kg}]$

Perfect Gas $\mathrm{c}_{\mathrm{p}}=$ constant $\left[\mathrm{kJ} / \mathrm{kg}{ }^{*} \mathrm{~K}\right]$

Kappa

$$
X=c_{p} / c_{v}[/]
$$

For air: $X=1.4[/], R=0.2867\left[\mathrm{~kJ} / \mathrm{kg}{ }^{*} \mathrm{~K}\right]$ and

$$
c_{p}=1.004\left[\mathrm{~kJ} / \mathrm{kg}^{*} \mathrm{~K}\right]
$$

# Compressitole Flow Engineering Eguations 

Sonic Velocity

$$
v_{s}=(X R T)^{1 / 2}[\mathrm{~m} / \mathrm{s}]
$$

Mach Number

$$
\mathrm{M}=\mathrm{v} / \mathrm{v}_{\mathrm{s}}[/]
$$

#  

$$
\begin{gathered}
\text { Energy Equation } \\
\mathrm{q}-\mathrm{w}=\left(\mathrm{h}+\mathrm{v}^{2} / 2+\mathrm{gh}\right)_{\text {out }- \text { in }}[\mathrm{kJ} / \mathrm{kg}]
\end{gathered}
$$

When $\mathrm{q}=0$ and $\mathrm{w}=0$ (for isentropic expansion), it follows:

$$
\left(h+v^{2} / 2+g h\right)_{\text {in }}=\left(h+v^{2} / 2+g h\right)_{\text {out }}
$$

## Furthermore,

in = Total and/or Stagnation Condition ( t ) and $\mathrm{v}=0$
out $=$ Static Condition
Also, gh = 0

# Compressitole Flow Engineering Eguations 

$$
\begin{gathered}
\text { Therefore, } \\
h_{t}=h+v^{2} / 2 \\
c_{p} T_{t}=c_{p} T+v^{2} / 2 \\
T_{t}=T+v^{2} /\left(c_{p} 2\right) \\
T_{t}=T\left(1+v^{2} /\left(T c_{p} 2\right)\right) \\
T_{t} / T=\left(1+v^{2} /\left(T_{p} 2\right)\right) \\
c_{p}-c_{v}=R \\
X=c_{p} / c_{v}
\end{gathered}
$$

## Compressible Flow Engineering Eguations

$$
\begin{gathered}
1-c_{v} / c_{p}=R / c_{p} \\
(X-1) / X=R / c_{p} \\
(X-1) /(X R)=1 / c_{p} \\
\text { Hence, } \\
T_{t} / T=\left(1+v^{2} /\left(T_{p} 2\right)\right) \\
T_{t} / T=\left(1+\left((X-1) v^{2}\right) /(T X R 2)\right) \\
v_{s}{ }^{2}=X R T
\end{gathered}
$$

## Compressionle Flow Engineering Eguations

$$
\begin{gathered}
T_{t} / T=\left(1+\left((X-1) v^{2}\right) /\left(v_{s}^{2} 2\right)\right) \\
M^{2}=\left(v / v_{s}\right)^{2} \\
T_{t} / T=\left(1+M^{2}(X-1) / 2\right)
\end{gathered}
$$

# Compressitole Flow Engineering Eguations 

Knowing the following:

$$
\begin{gathered}
T_{t} / T=\left(p_{t} / p\right)^{(X-1) / X} \\
p_{t} / p=\left(T_{t} / T\right) X(X-1) \\
p_{t} / p=\left(1+M^{2}(X-1) / 2\right)^{X^{\prime} /(X-1)}
\end{gathered}
$$

## Compressiogle Flow Engineering Eguations

Isentropic Flow

$$
\begin{aligned}
T_{t} / T & =\left(1+M^{2}(X-1) / 2\right)[/] \\
p_{t} / \mathrm{p}= & \left(1+M^{2}(X-1) / 2\right)^{X /}(X-1)[/] \\
h_{t} & =\left(h+v^{2} / 2\right)[k J / k g] \\
T_{t} & =\left(T+v^{2} /\left(2 c_{p}\right)\right)[K]
\end{aligned}
$$

Thrust $=v m+\left(p-p_{a}\right) A[N]$

