Engineering Software

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Engineering Assumptions

When dealing with energy conversion and considering ideal (isentropic) operation and the working fluid is air, the following assumptions are valid:

Compressible Flow

Single species consideration Basic equations hold (continuity, momentum and energy equations) Specific heat is constant Basic Engineering Equations

Basic Conservation Equations

Continuity Equation m = ρvA [kg/s]

Momentum Equation $F = (vm + pA)_{out - in} [N]$

Energy Equation q - w = $(h + v^2/2 + gh)_{out - in} [kJ/kg]$

Basic Engineering Equations

Ideal Gas State Equation pv = RT [kJ/kg]

Perfect Gas $c_p = constant [kJ/kg^K]$

> Kappa X = c_p/c_v [/]

For air: $\chi = 1.4$ [/], R = 0.2867 [kJ/kg*K] and $c_p = 1.004$ [kJ/kg*K]

Sonic Velocity $v_s = (\chi RT)^{1/2} [m/s]$

> Mach Number $M = v/v_s$ [/]

Energy Equation
q - w =
$$(h + v^2/2 + gh)_{out - in} [kJ/kg]$$

When q = 0 and w = 0 (for isentropic expansion), it follows:

$$(h + v^2/2 + gh)_{in} = (h + v^2/2 + gh)_{out}$$

Furthermore,

in = Total and/or Stagnation Condition (t) and v = 0

out = Static Condition

Also, gh = 0

Therefore, $h_{t} = h + v^{2}/2$ $c_{p}T_{t} = c_{p}T + v^{2}/2$ $T_{t} = T + v^{2}/(c_{p}^{2})$ $T_t = T(1 + v^2/(Tc_p 2))$ $T_t/T = (1 + v^2/(Tc_p^2))$ $c_{p} - c_{v} = R$ $X = C_{D}/C_{v}$

$$1 - c_v/c_p = R/c_p$$

$$(X - 1)/(X = R/c_p)$$

$$(X - 1)/(XR) = 1/c_p$$
Hence,

$$T_t/T = (1 + v^2/(Tc_p2))$$

$$T_t/T = (1 + ((X - 1)v^2)/(TXR2))$$

$$v_s^2 = X RT$$

$$T_t/T = (1 + ((X - 1)v^2)/(v_s^2 2))$$
$$M^2 = (v/v_s)^2$$

$$T_t/T = (1 + M^2(X - 1)/2)$$

Knowing the following:

 $T_t/T = (p_t/p)^{(X-1)/X}$ $p_t/p = (T_t/T)^{X/(X-1)}$

$$p_t/p = (1 + M^2(X - 1)/2)^{X/(X-1)}$$

Isentropic Flow $T_{t}/T = (1 + M^{2}(X - 1)/2)$ [/] $p_{t}/p = (1 + M^{2}(X - 1)/2)^{X/(X-1)}$ $h_{t} = (h + v^{2}/2) [kJ/kg]$ $T_t = (T + v^2/(2c_p)) [K]$ Thrust = $vm + (p - p_a)A[N]$